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Hélio Luiz Simonetti, Valério Silva Almeida, & Francisco de Assis das Neves

Federal Institute of Minas Gerais (IFMG)

ABSTRACT

This paper presents the Smoothing-ESO (SESO) technique for topology optimization of structures implemented in MATLAB using 4-node bilinear square elements. The lines comprising this code include definition of design domain, finite element analysis, sensitivity analysis, mesh-independency filter, optimization algorithm. Extensions and changes in the algorithm are also included in order to solve multiple load cases and compliant mechanisms design. In addition, a comparison is made with other optimization methods as Bi-directional Evolutionary Structural Optimization (BESO), Sequential Element Rejection and Admission (SERA) and Solid Isotropic Material with Penalization (SIMP). Thus, numerical examples are presented to demonstrate the ability of proposed methods to solve topology optimization problems.

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Classification: FOR Code: 290501

Language: English



LJP Copyright ID: 392833
Print ISSN: 2631-8474
Online ISSN: 2631-8482

London Journal of Engineering Research

Volume 19 | Issue 1 | Compilation 1.0



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ABSTRACT

This paper presents the Smoothing-ESO (SESO) technique for topology optimization of structures implemented in MATLAB using 4-node bilinear square elements. The lines comprising this code include definition of design domain, finite element analysis, sensitivity analysis, mesh-independency filter, optimization algorithm. Extensions and changes in the algorithm are also included in order to solve multiple load cases and compliant mechanisms design. In addition, a comparison is made with other optimization methods as Bi-directional Evolutionary Structural Optimization (BESO), Sequential Element Rejection and Admission (SERA) and Solid Isotropic Material with Penalization (SIMP). Thus, numerical examples are presented to demonstrate the ability of proposed methods to solve topology optimization problems.

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Author α: Department of Mathematics, Federal Institute of Minas Gerais (IFMG).

σ: School of Engineering of the University of São Paulo (EPUSP).

ρ: Department of Civil Engineering, Federal University of Ouro Preto (UFOP).

I. INTRODUCTION

Evolutionary structural optimization (ESO) method was firstly introduced by Xie and Steven (1993). The idea is based on a simple and empirical concept that a structure evolves towards an optimum by slowly removing elements with lowest stresses. Chu et al. (1996) to maximize the

stiffness of the structure, stress criterion was replaced with elemental strain energy criterion. Bi-directional evolutionary structural optimization (BESO) by Querin et al. (1998, 2000) method is an extension of that idea which allows for new elements to be added in the locations next to those elements with highest stresses. Rozvany and Querin proposed some improvements of this method under the term SERA (Sequential Element Rejection and Admission) where a “virtual material” was introduced, without the use of any intermediate densities or power law interpolations Rozvany and Querin (2004). Recently Matlab codes with extensions to Pareto strategies by Suresh (2010) and a methodology for ground structure based topology optimization in arbitrary 2D and 3D domains have been implemented using Matlab, Zegard and Paulino (2014, 2015). Three-dimensional topology optimization codes can be found in the work presented by Liu and Tovar (2014).

The present paper use of a Matlab program that incorporates the strategies for topology optimization based on the SESO technique. The proposed code is very similar to the 99-line by Sigmund (2001) except for the material update subroutine, where the optimality criterion has been replaced by the SESO algorithm. This program can be effectively used in personal computers for educational purposes of engineering students interested in the field of topology optimization, as an educational tool in courses on topology optimization.

II. FORMULATION AND OPTIMIZATION PROBLEM

2.1 Finite Element Model

The works of Sigmund (2001) and Andreassen et al. (2011) use the SIMP model to perform topology optimization considering the isotropic material. The SESO technique was implemented

in this article is based in the proposals by these authors. Thus, the calculations for isotropic material form are made in the finite element stiffness matrix formulation. The finite element used in this formulation is the 4-node bilinear square element. The stiffness matrix of this element results from the following double integration:

$$k_0 = t \int_{-1}^1 \int_{-1}^1 B^T D B J d\xi d\eta \quad \text{with } D = \frac{E}{1-\nu^2} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix} \quad (1)$$

In which: t is the thickness of the element. Since it is done in 2D, $t = 1$. D is the constitutive relation. E is the Young's modulus of the isotropic

material, ν is the Poisson's ratio of the isotropic material. The $[B]$ matrix is given as follows for this element:

$$[B] = \frac{1}{|J|} [B_1 \quad B_2 \quad B_3 \quad B_4] \quad \text{where } [B_i] = \begin{bmatrix} a \frac{\partial N_i}{\partial \xi} - b \frac{\partial N_i}{\partial \eta} & 0 \\ 0 & c \frac{\partial N_i}{\partial \eta} - d \frac{\partial N_i}{\partial \xi} \\ c \frac{\partial N_i}{\partial \eta} - d \frac{\partial N_i}{\partial \xi} & a \frac{\partial N_i}{\partial \xi} - b \frac{\partial N_i}{\partial \eta} \end{bmatrix} \quad (2)$$

The parameters a , b , c and d are given by:

$$\begin{aligned} a &= \frac{1}{4} [x_1(\xi-1) + x_2(-1-\xi) + y_3(1+\xi) + y_4(1-\xi)] & b &= \frac{1}{4} [x_1(\eta-1) + x_2(1-\eta) + y_3(1+\eta) + y_4(-1-\eta)] \\ c &= \frac{1}{4} [x_1(\eta-1) + x_2(1-\eta) + y_3(1+\eta) + y_4(-1-\eta)] & d &= \frac{1}{4} [x_1(\xi-1) + x_2(-1-\xi) + y_3(1+\xi) + y_4(1-\xi)] \end{aligned} \quad (3)$$

And the shape derivatives of N_i are given by:

$$\begin{aligned} N_1 &= \frac{1}{4} [(1-\xi)(1-\eta)] & N_2 &= \frac{1}{4} [(1+\xi)(1-\eta)] \\ N_3 &= \frac{1}{4} [(1+\xi)(1+\eta)] & N_4 &= \frac{1}{4} [(1-\xi)(1+\eta)] \end{aligned} \quad (4)$$

J is the determinant given by:

$$J = \frac{1}{8} \begin{bmatrix} x_1 & x_2 & x_3 & x_4 \end{bmatrix} \begin{bmatrix} 0 & 1-\eta & \eta-\xi & \xi-1 \\ \eta-1 & 0 & \xi+1 & -\xi-\eta \\ \xi-\eta & -\xi-1 & 0 & \eta+1 \\ 1-\xi & \xi+n & -\eta-1 & 0 \end{bmatrix} \begin{bmatrix} y_1 \\ y_2 \\ y_3 \\ y_4 \end{bmatrix} \quad (5)$$

2.2 Compliance Optimization Problem

The topology optimization problem for maximum stiffness structural design is defined as the

minimization of the compliance, where the objective is to find the material distribution that minimizes the structure's deformation under the

prescribed support and loading conditions, subjected to a volume constraint. In the SESO technique, a structure is optimized by removing $p\%$ and returning $(1-p\%)$ of the elements, that is, the element itself, instead of its associated physical or material parameters, is treated as the design variable. Therefore, to minimize compliance by removing elements, it is clear that the most effective way is to eliminate those elements having the lowest values so that the increase in compliance is minimal. Thus, taking into account the equilibrium, problem can be written as at equation 6. Where U is the global displacement, K is the global stiffness matrix, U_e and K_e are the element displacement vector and stiffness matrix, respectively, NE is the number of elements used to discretize the design domain, KU

$= F$ is the equilibrium equation F is the vector of the applied loads in the structure, x_i is the design variable of the i -th element and X is the vector of the design variables.

The global stiffness matrix K is assembled from the element stiffness matrices K_e , which are obtained multiplying the element isotropic stiffness matrix K_0 by x_i of the i -th element, since Young's moduli are assumed to depend linearly on the variable x_i , i.e., $K_e(x_i) = x_i K_0$. These design variables are discrete in the SESO technique, so variable x_i can only be zero or one. Nevertheless, in order to avoid obtaining a singular stiffness matrix, a non-zero lower bound is assigned to $x_i = 10^{-9}$ as shown in 6.

$$\begin{aligned}
 \text{Minimize } C &= \frac{1}{2} U^T K U = \frac{1}{2} \sum_{e=1}^{NE} U_e^T K_e U_e \\
 \text{sujeito a } & K U = F \\
 & V(X) = \sum_{i=1}^{NE} x_i V_i - \bar{V} \leq 0 \\
 & X = \{x_1 \quad x_2 \quad x_3 \quad \dots \quad x_N\}, \quad x_i = 1 \text{ ou } x_i = 10^{-9}
 \end{aligned} \tag{6}$$

III. COMPARING SESO WITH OTHER TOPOLOGY OPTIMIZATION METHODS WITH A MESH-INDEPENDENCY FILTER

The SESO algorithm will be compared to the ESO, SERA, Soft-Kill BESO and SIMP algorithms for this comparison it should be noted that the filter scheme is a heuristic technique for overcoming the checkerboard and mesh dependency problems in topology optimization. Therefore, it is better to compare the algorithms at two levels-one without

the mesh-independency filter and the other with it. A long cantilever shown in figure 1 is selected as a test example because it involves a series of bars broken during the evolutionary procedure of the topology optimization. A concentrated load $F = 1.0 \text{ N}$ is applied downward in the middle of the free end. Young's modulus $E = 1.0 \text{ MPa}$ and Poisson's ratio $\nu = 0.3$ are assumed. The design domain is discretized with 160×40 four node plane stress elements.

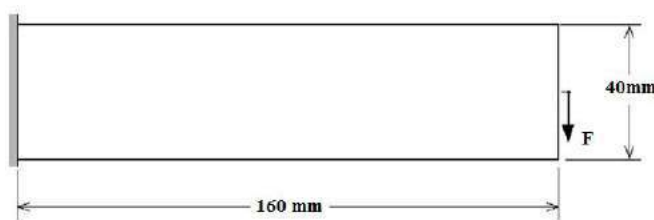

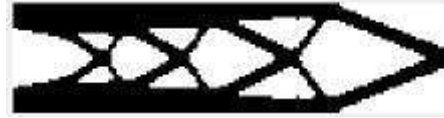




Figure 1: Design domain

Table 1 lists the used parameters and solutions obtained of the various topology optimization algorithms. It can be seen that the topology optimization algorithms produce very similar topologies except that the SIMP design has some

grey areas of intermediate material densities. The topologies presented in Table 1 present cleaner definitions of the members and are more conducive to practical use.

Table 1: Comparison of topology optimization methods with a mesh-independency filter

	Optimization parameters	Total iteration	Solutions	Compliance
SESO	ER=0.02	100		418,5
SERA	PR=0.03 SR=1.15	97		365.46
Soft-Kill BESO	ER=0.0125 P=3.0	80		196.41
SIMP	move=0.2 p=3	153		376.30

All presented methods show similar topologies. However, the topologies of the SESO, SIMP and soft-kill BESO methods are very similar. Although the soft-kill SESO converges with fewer iterations than SESO and SIMP, the lowest computational cost is due to SESO, however, SESO methods is the one that has the highest value for mean compliance in the optimal location analyzed.

VI. NUMERICAL EXAMPLES

4.1 EXAMPLE 1 - Compliant Mechanisms

The SESO method was extended for compliance minimization in solve compliant mechanisms topology optimization problems. A compliant mechanism optimum design involves two loading cases: input loading case and dummy (or adjoint) loading case. The allocation of force and displacement vectors for 'real' and adjoint load cases is similar to the two load case problem. Instead of calculating the compliance of the

structure, we will compute the output displacement, Saxena and Ananthasuresh (2000). Thus, sensitivities are obtained in terms of the solutions to the 'real' case and the adjoint loading case, which correspond to the first and second column of the displacement matrix U . The sign in the values of the sensitivities must be inverted as well, since we are trying to maximize the output displacement in this case, instead of minimizing the compliance. Figure 2a shows the design domain and the boundary conditions of the proposed problem. Figure 2b shows the optimal topology. The mesh employed consists of 40×20 elements and the filter radius equals 1.15 times the size of the finite element, with the volume constraint set to 30%. It noted that the convergence time and final topologies of compliant mechanisms are more sensitive to the values of the progression parameters than the maximum stiffness structures.

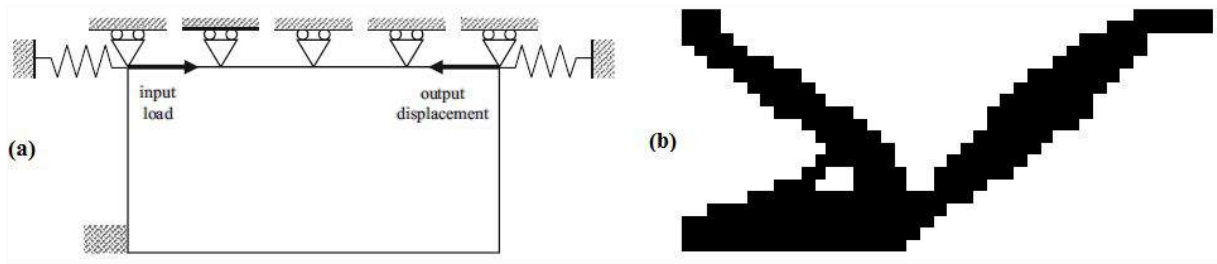


Figure 2: (a) Design domain and (b) Optimal topology

4.2 EXAMPLE 2 – MBB BEAM

The SESO method, with quadrilateral elements, was used to optimize the so-called MBB beam, which has been extensively studied in topology optimization, see Figure 3. The load is applied vertically in the upper left corner and there is a symmetric boundary condition along the left edge and the structure is supported horizontally in the

lower right corner. The beam's dimensions are 60×20 elements and correspond to half of the structure. The volume fraction limit is 50% and two different filter radii were used: 1.5 and 3.5. Figure 4a and 4b show the optimum configurations obtained with the present formulation.

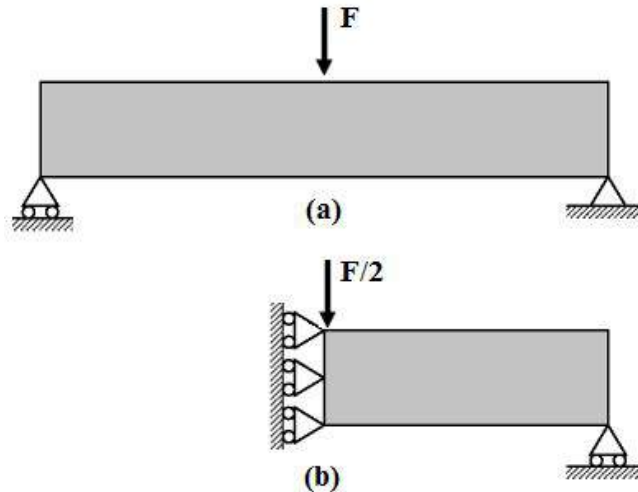


Figure 3: Design model for the topology optimization of the MBB-beam

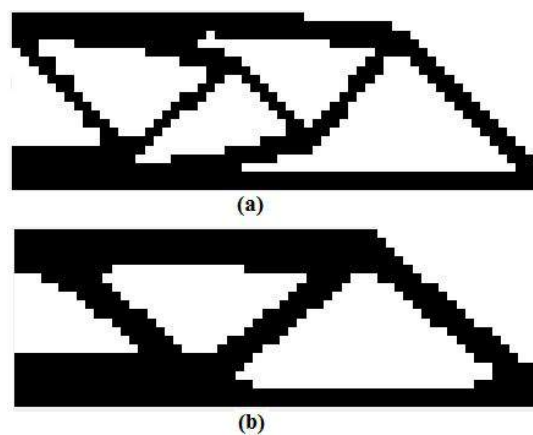


Figure 4: Topologies for the MBB-beam with different filtering
(a) Filter 1.5 and (b) Filter 3.5

4.3 EXAMPLE 3 – MULTIPLE LOAD CASES

The SESO method was extended for compliance minimization in solve multiple load cases topology optimization problems. If two load cases are considered, force and displacement vectors must become two-column vectors. The objective function should be defined as the sum of compliances for each load case. Figure 5a shows

the design domain and the boundary conditions of the proposed problem to cantilever beam. Figure 5b shows the optimum design for the cantilever when topology is optimized corresponds to the two-load case. It can be noticed that obtained topologies agree well with the solutions published in the paper by Sigmund (2001) using the SIMP method.

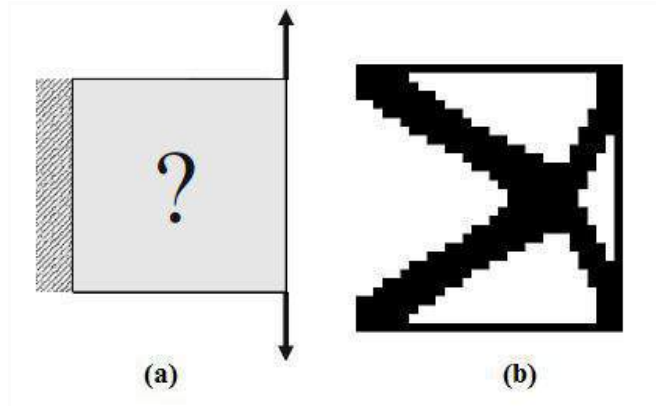


Figure 5: (a) Design domain and (b) Optimal topology

V. CONCLUSÕES

This paper presents SESO technique in code Matlab for minimum compliance problems and its extension to multiple load cases and compliant mechanism design. The code uses four-node bilinear square elements and the results presented with the numerical examples analyzed have shown that the SESO technique is efficient and robust in problems of plane elasticity.

ACKNOWLEDGEMENTS

The authors are grateful for the total support by grant #2016/02327-5, São Paulo Research Foundation (FAPESP) for the funding provided to the present research.

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